1. INTRODUCTION

The reserves of waste biomass from agricultural production in Serbia are very abundant. It is estimated that some 7000000 tons of this biomass is collected each year, with only a small portion of it being actually used for energy purposes. The total available energy potential of different types of waste biomass in Serbia is 135600 TJ/year, with straw and corn stems comprising a significant part – approximately 35500 TJ/year [1]. This biomass is a cheap and available fuel, but its utilization is linked to the problems of its collection, preparations for its transportation (cutting, tying into haystacks, baling), transportation and storage [2]. These problems are less significant and easier to handle if the biomass is used for energy purposes near places of its collection, in enterprises dealing with agricultural production, where the obtained energy – heat – could be used for heating of greenhouses, stables, poultry raising farms, offices, etc.

In the Agricultural Corporation PKB – Belgrade, one of the largest agricultural companies in Serbia, there are over 2000 ha of soya plantations. Each year after the harvest over 4000 tons of soya straw remains. Straw in general is a fuel of low calorific value [3]. This straw is baled, and is not used for energy purposes further on. At the same time, there is a need for heating of greenhouses inside the company, with the total greenhouse area to be heated equal to 5 ha. Until now, a technology for efficient combustion of baled material of this size has not been developed in Serbia. In the European Union, some technical solutions for solving these problems have been reported [4-6].

In the Laboratory for Thermal Engineering and Energy of the “Vinča” Institute in Belgrade, efforts have been made to develop a clean technology for utilizing baled biomass for energy production. As a result, an energy production facility – boiler (with thermal power of 1.5 MW) has been developed, for combustion of large soya straw bales. The first phase of the development was the building-up and testing of a real-scale demo energy production facility – furnace – with combustion of large rolled soya straw bales and with thermal power of 1 MW. The furnace was tested in order to examine the quality of combustion and to assess the possibilities for designing a soya straw-fired hot water boiler of similar characteristics. Since the results of these tests have proved to be very satisfactory, the energy facility has been designed and is being built at the moment. The results of the development and basic elements of the boiler design are presented here.

2. CONCEPT OF THE ENERGY FACILITY BURNING LARGE SOYA STRAW BALES

Soya straw bales, formed on the fields of PKB Corporation are square, with dimensions 0.7 x 1.2 x 2.7 m. Average low heating value is 13686 kJ/kg, and the density is 132.27 kg/m³. A scheme of the energy production facility burning the bales is given in Figure 1.

The fuel is fed to the furnace by the bale transporter (position 1 in Fig. 1). The combustion occurs in the furnace (pos. 2), made of refractory material – chamotte. Leaving the furnace chamber, the flue gases pass on to the gas-to-water heat exchanger (pos. 3), and then to the cyclone (pos. 4). The flue gases are run by the fan (pos. 5) and the smokestack (pos. 6). The boiler is equipped with a flue gas-water exchanger (pos. 3) placed downstream of the chamotte-lined furnace, and with screen barriers, multi-stage cyclones and scrubbers for the removal of particles from the flue gases. The boiler will be supplied with a system for automatic excess air control and with safety systems.
The boiler is equipped with a 100 m³ heat storage tank, which allows the whole facility to respond adequately and smoothly to the heating needs of the greenhouse, i.e. to weather changes.

A detailed scheme of the boiler combustion chamber is presented in Figure 2.

Bales of soya straw (position 1 in Fig. 2) are placed into the square cross section-shaped fuel feeding channel (pos. 2). The feeding of the bales is carried out continuously, using the bale transporter (pos. 2) by which the bales are slowly pushed towards the inside of the furnace. The feeding is adjusted according to the thermal power to be achieved, and therefore produced thermal power output of the facility should be stable.

The combustion of the bales is based on the principles of combustion of a cigarette – only the portion of the bale, placed in the furnace itself (the bale forehead), combusts at a time. As the bale forehead is burnt out, the following portion of the bale enters the combustion zone (4) inside the furnace, by means of fuel feeding. This portion had been exposed to devolatilization (5) prior to entering the furnace, due to the heat absorbed from the combustion zone. The portion of the bale placed inside the feeding tube, near the entry point of the feeding tube into the furnace, is exposed to drying (6).

The bale forehead is supported from the bottom by the water cooled grate (7). The combustion air is supplied in three stages. The primary air (8) is supplied through:

- Eight nozzles placed around the bale, on its perimeter, at the feeding channel entry point into the furnace (8a);
- Two nozzles placed under the grate (8b).

The nozzles have been dimensioned in such a way that primary air supply on the bale perimeter is practically uniform.

The secondary air (9) is supplied through nozzles placed in a movable (both along the bale axis and rotating around the axis) secondary air supplier (10), which is also used for shaking off of the burnt out ash from the bale forehead, and for exact positioning of the bale inside the furnace according to thermal power requirements. This element is water cooled, and run by an engine. The tertiary air (11) is supplied through nozzles placed in the flow barriers in the convection section, just above the furnace. The flow barriers have been designed in order to cool the flue gases and to complete the combustion process as much as possible before the furnace exit.

The air is supplied with two fans, one for primary air supply, and the other, a smaller one, for secondary and tertiary air. The flue gases exit the facility through the opening (14).

3. THE DEMO FURNACE BURNING SOYA STRAW BALES

Before the actual boiler has been built, in order to assess the combustion quality and to obtain data for the design of the soya straw-fired hot water boiler, a demo furnace had been designed and built. The appearance of the furnace is shown in Figure 3. This furnace was adopted for cylindrical bales, with 1.2 m in diameter.
The inside of the furnace, with the installed thermocouple probe, the movable cross and the grate is shown in Figure 4.

The sum of five tests was done. A summary of main test parameters is given in Table 2.

<table>
<thead>
<tr>
<th>Test parameters</th>
<th>1</th>
<th>2</th>
<th>3</th>
<th>4</th>
<th>5</th>
</tr>
</thead>
<tbody>
<tr>
<td>Number of bales in the feeding tube</td>
<td>1</td>
<td>2</td>
<td>2</td>
<td>3</td>
<td>3</td>
</tr>
<tr>
<td>Amount of straw [kg]</td>
<td>134.6</td>
<td>280</td>
<td>327.97</td>
<td>458.3</td>
<td>554.9</td>
</tr>
<tr>
<td>Primary air [m³/h]</td>
<td>1548</td>
<td>1548</td>
<td>1548</td>
<td>1350</td>
<td>1404**</td>
</tr>
<tr>
<td>Secondary air [m³/h]</td>
<td>-</td>
<td>-</td>
<td>234</td>
<td>418.25</td>
<td>228.14**</td>
</tr>
<tr>
<td>Tertiary air [m³/h]</td>
<td>504</td>
<td>252</td>
<td>108</td>
<td>259.2</td>
<td>259.2**</td>
</tr>
<tr>
<td>Calculated thermal power [kW]</td>
<td>485.2</td>
<td>529.3</td>
<td>556.5</td>
<td>551.7</td>
<td>455.7</td>
</tr>
<tr>
<td>Average air excess coefficient, λ</td>
<td>not measured</td>
<td>4.71</td>
<td>2.61</td>
<td>4.12*</td>
<td>2.92**</td>
</tr>
<tr>
<td>Test duration [min]</td>
<td>47</td>
<td>89</td>
<td>99</td>
<td>140</td>
<td>205</td>
</tr>
</tbody>
</table>

Conditions: * – The thermal power was calculated over the entire test period; ** – The air flow rates refer only to the period shown in the diagrams (Figs. 10 and 11), since in Test 5 variable speed drives were used for changing the speed of the fans. The air excess coefficient λ was calculated for the same period.

During all tests, three temperatures were measured, with shielded type K thermocouple probes. Thermocouple probe locations, as well as the location of the gas sampling probe, are shown in Figure 5. Gas sampling was done with a probe placed near the furnace exit (Fig. 5). Gas samples were continuously analyzed with two analyzers, collected every 5 seconds and stored on-line.

It should be noted that secondary air supply through the movable cross was not present in the first version of the furnace, which was examined in Tests 1 and 2. The results from these tests stressed the need to introduce secondary air in the combustion zone, at the bale forehead, and the furnace with secondary air supply was examined in Tests 3, 4 and 5.

Test 1 was conducted with one bale of straw placed in the feeding tube. Only temperature measurements were done, and the results showed that the temperature in the combustion zone, in steady conditions, was quite stable (730 – 830 °C, Fig. 6) for a reasonable period of time (40 minutes). It was noted that the amount of tertiary air did not contribute much to overall combustion conditions, and that in fact this air over-cooled the flue gases in the combustion zone.

In Test 2, along with temperatures, gas composition was continuously measured. Less air was supplied as tertiary than in Test 1. In the initial, start-up period (Fig. 7), gas samples were taken directly from the combustion zone, and very high levels of CO in the flue gases were noted. After the choking of the gas sampling probe and its cleaning, and also in all following tests, gas samples were taken only from the top of the furnace, as it was shown in Figure 5. As the temperature in this period increased to approximately 1000 °C, bale feeding was slowed down, which corresponds to the temperature downfall (minutes 50 – 75). Soon after that, stable conditions were obtained (Figs. 7 and 8), primarily by adjusting bale feeding.

High level of CO concentration at the furnace top in Test 2 urged the introduction of a small amount (approximately 10 % of total air) of secondary air in the

![Figure 4. Inside the furnace – the thermocouple probe, the movable secondary air supplier used for secondary air supply and shaking-off of the ash, and the grate](image_url)
combustion zone, which would cool down the movable cross at the same time. It was also noted that tertiary air flow rate should be decreased, and therefore secondary air was introduced to the detriment of tertiary air. This change in design was examined in Test 3, with two bales placed in the feeding tube.

![Figure 7. Test 2 – Temperature in the combustion zone vs. CO concentration](image)

![Figure 8. Test 2 – O2 concentration](image)

The principal aim of Test 4 was to assess the possibility of longer furnace operation, with three bales placed inside the feeding tube. The bales prepared for this test were approximately 1.2 m in diameter, and in order to secure stable manual feeding, the gaps between the tube wall and the bales were manually filled with more straw. Problems with feeding undersized bales caused instabilities in the first hour of the test. In the period shown in Figures 11 and 12, the temperature was in the desired range, and CO concentration was acceptable for most of the time (up to 350 ppm), the only rise in CO occurring at the time of the temperature downfall (minutes 100 – 110). It was spotted by visual inspection, through the inspection openings, that the bale was not inside the furnace at the time of the downfall, due to the problems with manual bale feeding and cross positioning – the bale forehead remained inside the tube. This caused the flame to enter the tube at the time, which also occurred during Test 5.

The supply of the secondary air through the cross provided excellent conditions for combustion completion (Fig. 9) – the concentration of CO was equal to zero for most of the time during the test. The air distribution (82 % primary air, 12 % secondary, 6 % tertiary) was found to be well suited for maintaining steady conditions inside the furnace. On the other hand, the stability of the thermal output was found to depend largely on the active length of the bale immersed into the furnace. Therefore, it is of great importance to feed the bale uniformly in accordance with the combustion process, and to maintain this length as stable as possible, by moving the bale holder, used also for secondary air supply, accordingly. The temperature instabilities (from the minute 45 further on, Fig. 9) during this test are a consequence of changes of this length. The only peak in CO concentration coincided expectedly with low temperatures during this period. Nevertheless, this test proved that the adopted concept of the furnace provided good conditions for efficient combustion of soya straw bales, with O2 concentration ranging from 10 – 14 % (Fig. 10), and an optimal average value of $\lambda$.

![Figure 9. Test 3 – Temperature in the combustion zone vs. CO concentration](image)

![Figure 10. Test 3 – O2 concentration](image)

The aim of Test 5 was to assess the influence of air flow rate control, with variable speed drives, on furnace performance. During a chosen period of 40 minutes (Figs. 13 and 14), optimal air flow rates were obtained and bale feeding was kept stable. The concentration of CO was very low, with O2 concentration varying in the range of 10 – 15 %. The temperature during this period was higher than the desired 850 °C (which should not be exceeded in order to avoid ash melting), which will be taken into consideration in some of the conclusions.
4. CONCLUSION

The boiler concept for combustion of large soya straw bales has been developed. In order to assess the combustion quality and to obtain data for the design of a soya straw-fired hot water boiler, a demo furnace has been designed, built and tested. Some conclusions, based on test results and on noted phenomena during the tests of the experimental facility, can be drawn.

Bale dimensions should be as uniform as possible and in accordance with the feeding channel diameter, in order to fill up the tube with fuel, which makes bale feeding easier. Bale quality should also be uniform, with respect to density and humidity. The bales should be stored in a way that keeps their form, and should be handled mechanically, by being pushed with hydraulic systems. One part of the feeding tube could be placed outside the boiler. Based on the appearance of the feeding tube near the furnace, which together with the results suggested that fire entered the tube, at least two bales must be in the process all the time, and this part of the feeding tube must be water-cooled. Good sealing must be secured in the feeding section as a whole.

In order to avoid unstable operation, the bale should “peek” over the furnace space very little during start-up. Start-up and achieving steady state should be done manually, and after that, steady operation should be maintained by automatic control.

Thermal power control is achieved by bale feeding control and by control of the active bale length inside the furnace. The control of the maximum temperature inside the furnace is done with the secondary air stream flowing towards the bale forehead. The rotation of the movable cross is necessary, for ash shaking-off and better distribution of secondary air. All elements for secondary air introduction must be well cooled with water. The tertiary air flow rate should be as low as possible.

Since the role of the secondary and the tertiary air is mainly to prevent the occurrence of high temperatures, a possibility of introduction of the re-circulated flue gas instead of secondary air should be taken into consideration. Although during all tests the temperature increased to over 850 °C at certain times, there were no problems with ash melting and deposition by sticking.

ACKNOWLEDGEMENTS

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REFERENCES


**NOMENCLATURE**

\( T_1 \) temperature in the combustion zone \(^{[\circ C]}\)

\( T_2 \) temperature in the zone above the bale \(^{[\circ C]}\)

\( T_3 \) exit flue gas temperature \(^{[\circ C]}\)

**Greek symbols**

\( \lambda \) air excess coefficient

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КОТАО ЗА САГОРЕВАЊЕ ВЕЛИКИХ БАЛА
СОЈИНЕ СЛАМЕ ЗА ГРЕЈАЊЕ ПЛАСТЕНИКА

Растко Младеновић, Милијана Паприка, Мирко Коматина, Александар Ерић, Милица Младеновић, Драгољуб Дакић

У једној од највећих компанија које се баве пољопривредном производњом у Србији под засадом соје се налази преко 2000 ha, и сваке године се произведе око 4000 t балиране сојине сламе. Планирано је да се сојина слама користи за грејање пластеника, укупне површине 5 ha. Због тога, у Лабораторији за термотехнику и енергетику Института за нуклеарне науке „Винча“ је учињен напор да се развије технологија за коришћење крупне балиране биомасе за производњу енергије. У првој фази, развијено је демонстрационо- eksperimentalno постројење – ложиште за сагоревање бала сојине сламе. Ложиште је изградљено и термотехнички испитано, да би се утврдио квалитет сагоревања. Како су резултати испитивања били веома задовољавајући, у другој фази развоја, врело водон котао сличних карактеристика (који сагорева бале сојине сламе димензија 0,7 x 1,2 x 2,7 m) је пројектован, и тренутно је у фази изградње.